| Scheme of Valuation/Answer Key <br> (Scheme of evaluation (marks in brackets) and answers of problems/key) |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| APJ ABDUL KALAM TECHNOLOGICAL UNIVERSITY SIXTH SEMESTER B.TECH DEGREE EXAMINATION, APRIL 2019 |  |  |  |  |  |
| Course Code: ME304 |  |  |  |  |  |
| Course Name: DYNAMICS OF MACHINERY |  |  |  |  |  |
| Max. Marks: 100 |  |  |  | Duration: 3 Hours |  |
| PART A |  |  |  |  |  |
|  |  | Answer any | three full questions, each carries 10 | arks. | Marks |
| 1 | a) | Configuration diagram <br> Free body diagram of li <br> Torque $=878 \mathrm{Nm}$ (coun |  -3 marks <br> ks -6 marks <br> clockwise) -1 marks |  | (10) |
| 2 |  | Graphical Method <br> Configuration diagram Vector diagram of link Torque <br> Analytical Method: <br> Free body diagram/Con <br> Equilibrium equations <br> Torque - 2 <br> (Graphical/analytical/ <br> the problem) | irtual work/matrix method can be | used for so | (10) |
| 3 | a) | Configuration diagram <br> Inertia force $=-4236$ <br> Inertia torque due to rec <br> Correction torque $=45$. <br> Torque due to weight of <br> Total inertia torque on | procating parts $=-248 \mathrm{Nm}$ <br> 7 Nm <br> mass $=-27.14 \mathrm{Nm}$ counter clockwise <br> e crank shaft $=320.2$ clockwise | -3 marks <br> - 1 mark <br> - 1 mark <br> - 2 mark <br> - 2 mark <br> - 1 mark | (10) |
| 4 | a) | $\begin{array}{ll} \hline \text { Statement }-2 \text { marks } \\ \text { Explanation }-3 \text { marks } \end{array}$ |  |  | (5) |


|  | b) | Figure - 2 marks <br> Explanation - 3 marks | (5) |
| :---: | :---: | :---: | :---: |
| PART B |  |  |  |
| Answer any three full questions, each carries 10 marks. |  |  |  |
| 5 |  | Turning moment diagram -3 marks <br> Resultant turning moment diagram -2 marks <br> Power developed $=4.24 \mathrm{KW}$ -1 marks <br> Maximum fluctuation of energy $=11.78 \mathrm{Nm}$ -1 marks <br> Coefficient of fluctuation of speed $=0.04$ or $4 \%$ -1 marks <br> Coefficient of fluctuation of energy $=0.0278$ or $2.78 \%$ -1 marks <br> Maximum angular acceleration of the flywheel $=292 \mathrm{rad} / \mathrm{s}^{2}$ -1 marks | (10) |
| 6 | a) | Table - -4 marks  <br> Primary crank and secondary crank position -2 mark  <br> Couple polygon -2 marks  <br> Force polygon -2 marks  | (10) |
| 7 |  | Gyroscopic couple -2 marks  <br> Centrifugal couple -2 marks  <br> Total over turning couple -2 marks  <br> Balancing couple -2 marks  <br> Angle of heel -2 marks  <br> (Wheel radius (Rw) was not given in the question. So, a suitable value can be assumed or an expression with Rw can be formulated. Marks in proportion to the number of steps can be given.) | (10) |
| 8 |  | Figure - 1 marks <br> Gyroscopic couple due to four wheels $=37.1 \mathrm{Nm} \quad-2$ marks <br> Gyroscopic couple due to rotating parts of the engine $=34.7 \mathrm{Nm}-1$ marks <br> Centrifugal force $=9263 \mathrm{~N} \quad-1$ marks <br> Centrifugal couple $=4631.5 \mathrm{Nm} \quad-1$ marks <br> Load on the front wheel $1=4322.86 \mathrm{~N} \quad-1$ marks <br> Load on the front wheel $2=7435.26 \mathrm{~N}-1$ marks | (10) |


|  |  | Load on the rear wheel $3=2374.74 \mathrm{~N} \quad-1$ marks <br> Load on the rear wheel $4=5487.14 \mathrm{~N} \quad-1$ marks |  |
| :---: | :---: | :---: | :---: |
| PART C |  |  |  |
| Answer any four full questions, each carries 10 marks. |  |  |  |
| 9 | a) | Explanation - 2 marks | (2) |
|  | b) | 1. Stiffness of the spring $=877 \mathrm{~N} / \mathrm{m}$ -2 marks <br> 2. Logarithmic decrement $=0.278$ -2 marks <br> 3. Damping factor $=0.0442$ -2 marks <br> 4. Damping coefficient $=7.4 \mathrm{~N} / \mathrm{m} / \mathrm{s}$ -2 marks | (8) |
| 10 | a) | 1. Stiffness of each spring $=49368 \mathrm{~N} / \mathrm{m}$ -4 marks <br> 2. Dynamic force transmitted $=39.27 \mathrm{~N}$ -3 marks <br> 3. Natural frequency of the system $=45.35 \mathrm{~Hz}$ -3 marks | (10) |
| 11 | a) | Figure - 1 mark <br> Explanation of term dynamic magnifier - 2 marks <br> Explanation of term transmissibility - 2 marks | (5) |
|  | b) | It can be solved with the given data. Moment of inertia as $10^{9}$ or $109 \mathrm{~mm}^{4}$ and E as $205 \times 103$ or $205 \times 10^{3} \mathrm{~N} / \mathrm{mm}^{2}$ can be used. Deflection values may not be reasonable as the given data are not correct. Marks shall be given if the students use correct data and get different answers. | (5) |
| 12 |  | Figure of torsionally equivalent shaft -4 marks <br> 1. Diameter 'd' mm for the shaft $\mathrm{CD}=91.7 \mathrm{~mm}$ -3 mark <br> 2. Natural frequency of free torsional vibration $=3.33 \mathrm{~Hz}$ -3 marks | (10) |
| 13 | a) | Explanation of term whirling speed of a shaft -2 marks proof - 3 marks | (5) |
|  | b) | Figure of shaft -1 marksStatic deflection due to 1 kg of mass at the centre $=28 \times 10^{-6} \mathrm{~m}-1$ marksStatic deflection due to mass of the shaft $=0.133 \times 10^{-3} \mathrm{~m}$Frequency of transverse vibration $=43.3 \mathrm{~Hz}$ -1 marks <br> Whirling speed of a shaft $=2598 \mathrm{rpm}$. -1 marks | (5) |


| 14 | a) | Node lengths $1_{\mathrm{A}}=1.146 \mathrm{~m}$ OR 0.4356 $\mathrm{l}_{\mathrm{C}}=1.91 \mathrm{~m} \text { OR } 0.726 \mathrm{~m}$ <br> Figure <br> Frequency $1=171 \mathrm{~Hz}$ <br> Frequency $2=277 \mathrm{~Hz}$ | - 2 marks <br> - 2 marks <br> - 4 mark <br> - 1 mark <br> - 1 mark | (10) |
| :---: | :---: | :---: | :---: | :---: |
| **** |  |  |  |  |



